The power output to the USB port of this device is 5 W, the voltage is 5V. The weight of the device is 8.16 kg. The cooking surface area is 50.5 cm², the diameter of the device is 33 cm. The cost of the generator is 301 US dollars.

The disadvantage of this design is the significant weight and low efficiency of thermoelectric conversion. The ratio of electric power to the weight of the generator with the stove is 0.6.

The thermoelectric generator **FireBee Power Tower** [2] converts heat from any portable stove into electricity for charging smartphones, tablets and other electronic gadgets.

The device can be used with various heat sources, it can achieve an electrical power of 10 W at a voltage of 5V, but its disadvantage is that in addition to the heat of the stove, for its operation it needs a regular replacement of the heated water with cool water. This creates an inconvenience in the field and makes it much more difficult to use this device.

The thermoelectric generator [3] comprises thermoelectric generator modules, "hot" heat exchangers, "cold" heat exchangers. "Hot" heat exchangers are immersed in the reservoir of a hot geyser, and "cold" heat exchangers are buried in the "permafrost" or immersed in a cold reservoir. The thermoelectric generator works as follows. Hot "heat exchangers" are heated from the hot reservoir of the geyser and supply heat to the thermoelectric generator modules, while "cold" heat exchangers remove heat from the thermoelectric generator modules and are cooled in "permafrost" or in a cold reservoir. Due to the temperature difference created by "hot" and "cold" heat exchangers, thermoelectric generate electricity. Thus, for the operation of a thermoelectric generator, natural sources of heating and cooling are used. This design solution in the actual operation of the device requires the presence of natural sources of heat and cold. This fact makes impossible wide application of such a device.

The tourist generator PowerSpot Mini Thermixc [4] realizes a stable output power of 7 W and allows charging electronic devices in the appropriate time:

Mobile phone (1500 mAh) - 1 h 30 min

Smartphone (3000 mAh) - 3 h

iPhone 6 (1800 mAh) - 1 h 45 min

iPhone 7 (1969 mAh) - 2 h

iPhone 7 plus (2900 mAh) - 2 h 50 min

iPad / tablet (6500 mAh) - 6 h 30 min

GoPro HER04 (1160 mAh) - 1 h 10 min

The developers declare a service life of 50.000 hours at operating temperatures of 150 °C - 400 °C. For operation, the device consumes about 50 g of liquefied gas. This circumstance makes regular use of the generator in the field practically impossible.

The purpose of this work is to create and study a highly efficient portable thermoelectric generator, which would have low weight and size and would be economically available to a wide range of consumers.

Physical model of a TEG with a heat source

Fig. 1 shows a physical model of thermoelectric generator unit comprising a thermopile, heat spreaders for heat supply and removal from the thermopile, a device for intensive heat removal and a heat source – flat-parallel surface uniformly heated with flame.

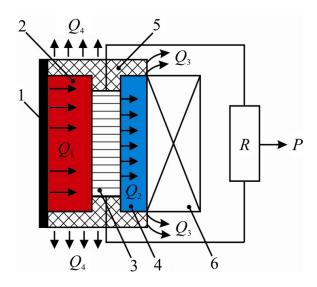


Fig. 1. Physical model of thermoelectric generator unit:
1 – heated surface; 2 – hot heat spreader;
3 – thermopile; 4 – cold heat spreader;
5 – housing; 6 – thermopile cooling block.

Since the generator is built into a heated surface, the processes of heat exchange between a real source of fuel combustion and this surface are not considered. The temperature of the heated surface is assumed to be equal to the temperature of the hot TEG heat exchanger.

Thus, heat supply from the heated surface to the hot side of the thermopile and heat removal from the thermopile cold junctions to the cold heat exchanger is carried out due to thermal conductivity and is described by the equations [5]:

$$Q_1 = \frac{\lambda_T S_T}{l_T} (T_T - T_T), \qquad (1)$$

$$Q_2 = \frac{\lambda_m S_m}{l_m} (T_X - T_m),, \qquad (2)$$

where $\lambda_{T_r} \lambda_m$ is thermal conductivity of material of the hot and cold heat conductors; T_T , T_m are temperatures of the hot and cold heat conductors; T_T , T_X is temperature of the hot and cold side of thermopile, respectively.

Thermal power Q3 is removed from the cold heat conductor by free convection into the water contained in the cooling block (pot capacity):

$$Q_3 = \alpha (T_m - T_0) S_m , \qquad (3)$$

where $\alpha(v)$ is the coefficient of convective heat transfer between the cold heat conductor and water in the cooling block; T_0 is the temperature of the liquid in the cooling block.

The electric power generated by the thermopile is proportional to Q_1 and the efficiency of the thermopile η :

$$P = P_{TEE} = Q_1 \eta , \qquad (4)$$

The main heat losses Q_4 occur on the thermopile through thermal insulation:

$$Q_4 = \frac{\lambda S_T}{L} (T_{\mathcal{B}} - T_0) , \qquad (5)$$

where λ is the thermal conductivity of the insulating material; S_T is the surface area of the hot heat conductor which is not occupied by the thermopile; L is the thickness of the insulating layer.

The heat balance equation for the selected model of the thermoelectric generator can be written as:

$$\begin{cases} Q_1 = P + Q_2 + Q_4, \\ Q_2 = Q_3 + Q_4 \end{cases}$$
(6)

The solution of the system of equations (6) makes it possible to determine the main energy and design parameters of the thermoelectric generator unit in particular and a complex unit with a portable heater in general.

Optimization of TEG design

Optimization of the generator unit was preceded by an experiment to determine the temperatures of the elements of the selected portable stove [6]. Fig. 2 presents the results of such measurements.

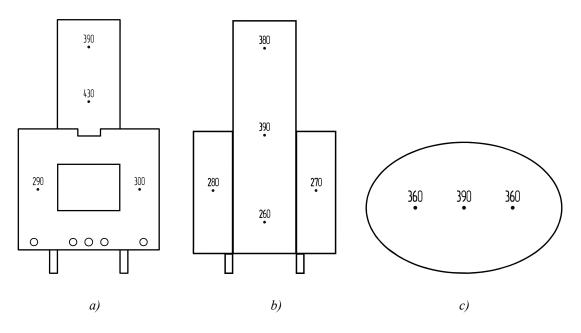


Fig.2. Results of measuring the temperatures of the heater walls (°C)
a) - front view, b) - rear view, c) - top view.

Optimization computer calculations, which took into account the experimental temperature measurements, made it possible to determine the design parameters of the thermoelectric generator unit which was designed to be placed on the cooking surface of a portable stove.

From the computer analysis it followed that thermoelectric generator unit based on a military pot should contain two thermoelectric generator modules in its bottom facing the heat source. The

Altec-1061 module is installed as the optimal thermoelectric module for certain temperature and thermal conditions.

Calculation of energy characteristics of the TEG with a stove

The approximate calculated mass of firewood at one loading in the stove m = 60g = 0.06 kg. When burning one load of firewood, the released energy *E* is:

$$E = G \cdot m = 750 \text{ (kJ)} \tag{7}$$

where G=12.56 mJ/kg is calorific value of firewood.

Thermal power Q absorbed by thermoelectric modules:

$$\eta = \frac{\mathbb{P}}{\mathbb{Q}} \longrightarrow Q = \frac{\mathbb{P}}{\eta} = 110 \text{ (W)}$$
(8)

where P = 5 W is calculated electric power generated by modules, $\eta = 0.045$ is the efficiency of modules at the hot and cold side temperatures $T_h = 300^{\circ}$ C and $T_c = 100^{\circ}$ C, respectively.

Operating time *t* at one loading while minimizing heat losses:

$$Q = \frac{E}{t} \rightarrow t = \frac{E}{Q} = 2 \text{ (h)}$$
(9)

Thermal power $Q_{\rm H}$ consumed to heat water in the generator pot:

$$Q_{\mu} = Q - P = 105 \,(\mathrm{W}) \tag{10}$$

Time *t* of heating water in the generator pot:

$$Q_{\rm H} = \frac{c \cdot m \cdot (T_1 - T_0)}{t} \to t = \frac{c \cdot m \cdot (T_1 - T_0)}{Q_{\rm H}} = 1 \,({\rm h}\,) \tag{11}$$

where $c=4.22 \ kJ / kg \cdot K$ is heat capacity of water;

m=11 is the volume of water in the pot;

 T_1 =100 °C is the final water heating temperature;

 $T_2=20$ °C is the initial temperature of water in the pot.

In the absence of heat loss, the operating time of the thermoelectric generator at one loading of fuel can be approximately 2 hours.

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$$Q = \frac{E}{t} \rightarrow t = \frac{E}{Q} = 2 \text{ (h)}$$
(9)

Thermal power Q_{H} consumed to heat water in the generator pot:

$$Q_{\rm H} = Q - P = 105 \,(\rm W) \tag{10}$$

Time *t* of heating water in the generator pot:

$$Q_{\rm H} = \frac{c \cdot m \cdot (T_1 - T_0)}{t} \to t = \frac{c \cdot m \cdot (T_1 - T_0)}{Q_{\rm H}} = 1 \,({\rm h})$$
(11)

where $c=4.22 \ kJ / kg \cdot K$ is heat capacity of water;

m=11 is the volume of water in the pot;

 T_1 =100 °C is the final water heating temperature;

 $T_2=20$ °C is the initial temperature of water in the pot.

In the absence of heat loss, the operating time of the thermoelectric generator at one loading of fuel can be approximately 2 hours.

Description of TEG design

The design of a thermoelectric unit for work with a portable stove is shown schematically in Fig. 3.

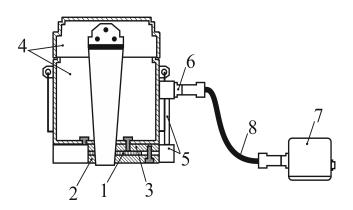


Fig.3. Schematic of thermoelectric generator unit.1 – thermopile; 2 – heat-conducting plate; 3 – heat sink plate; 4 – army pot with a lid; 5 – protective housing; 6 – electric output; 7 – electronic voltage stabilization unit; 8 – electric connecting cable. To protect the electrical terminals of the thermopile from direct flame and external mechanical loads, the generator contains a protective housing **5**, which ends with an electrical output **6**. Using an electric cable **8**, the thermoelectric generator is connected to the electronic output voltage stabilization unit **7**. The appearance of the unit is shown in Fig. 4.



Fig. 4. Appearance of the thermoelectric unit

The Institute of Thermoelectricity of the National Academy of Sciences and the Ministry of Education and Science of Ukraine has developed, researched and standardized a thermoelectric unit for universal use with various heat sources and fuels. Table 1 shows the main parameters of the Altec - 8046 unit [7].

<u>Table 1</u>

1	Electric power, W	5
2	Electric voltage output, V	5.10
3	Pot volume, l	1.3
4	Overall dimensions, mm	$170 \times 170 \times 100$
5	Mass, kg	1

Basic parameters of the Altec-8046 thermoelectric unit

Methods of experimental research

The purpose of research conducted at the Institute of Thermoelectricity was to determine the energy characteristics of a thermoelectric army pot on a portable stove. The maximum electric power of the generator was measured in the range of water temperatures $T_w = (20-100)$ °C every 10°C from the moment of ignition of the stove. The schematic of the experiment is shown in Fig. 5.

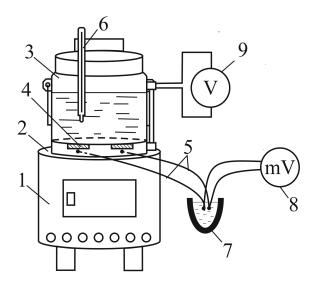


Fig.5 Schematic of the experiment to study the energy characteristics of thermoelectric army pot.
1 - tourist stove; 2 - cooking surface; 3 - thermoelectric generator;
4 - thermoelectric generator modules; 5 - thermocouples;
6 - thermometer; 7 - vessel with melting ice;8 - millivoltmeter; 9 - voltmeter.

When studying the energy characteristics of the generator, at all stages of the experiment, the fuel consumption was recorded to determine the obtained thermal power and the efficiency of thermoelectric conversion.

Research results

The time dependences of the energy characteristics of the Altec-8046 thermoelectric unit with a portable stove are presented in Fig. 6.

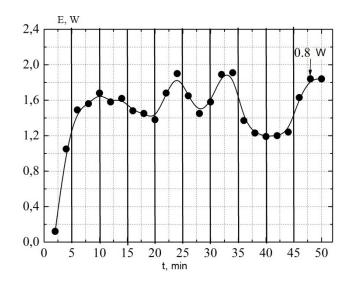


Fig. 6. Dependence of electromotive force E of thermoelectric modules on time t. Vertical black lines indicate the moments of throwing fuel into the stove.

In these studies, firewood was used as fuel. Firewood consumption g = 840 g / hour. Thermal design power of the stove Q = 2.9 kW. For comparison, a study of a thermoelectric generator unit was carried out on an open flame from dry alcohol. Fig. 7 shows the obtained dependence of the electromotive force and the value of power on time.

The comparison of the obtained results showed the expediency of refining the portable stove, which would allow operation of the thermoelectric unit with an open flame. This design solution can improve the efficiency of TEG by a factor of ~ 1.6 .

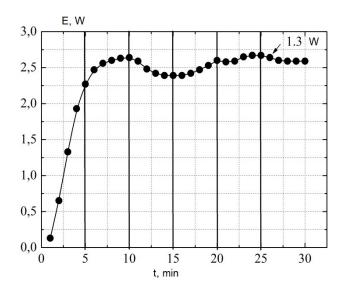


Fig. 7. Dependence of electromotive force on time in the variant of open flame

The consumption of dry alcohol g = 420 g / h. In this case, the thermal power of the stove was Q = 3.6 kW. The volume of water poured into the pot is 1 liter. The achieved efficiency values were about 1% for a TEG with a portable stove. The ratio of the output power to the weight of the device with a wood-fired stove is ~ 0.8, with an open flame - 1.3. These values are higher than those of the closest analogues.

Economic calculations of the cost of the developed device were carried out. Table 2 presents the cost of a single product of a thermoelectric generator with an army pot "Altec-8046" versus the batch size.

Table 2

The cost of thermoelectric generator versus the batch size

Batch size, pcs.	1	10	100	1000
Cost, \$	190	178	163	150

Conclusions

- 1. A thermoelectric generator based on the Altec-8046 thermoelectric unit with a portable stove has been developed.
- 2. Studies carried out on various fuels have shown the possibility of using the developed device for power supply of modern means of communications and various gadgets.

- 3. The achieved values of the output electric power with respect to the weight of the device significantly outweigh the closest known analogues.
- 4. The expediency of constructive revision of the selected portable stove in terms of providing the possibility of using an open flame has been established.
- 5. The energy efficiency of a TEG with a portable stove after its improvement can increase by a factor of 1.6.
- 6. The economic calculations of the device have determined the average cost of the TEG as \$170. The authors express their sincere gratitude to the scientific supervisor, academician of the

NAS of Ukraine LI Anatychuk, for the idea of work and valuable advice in its implementation.

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ТЕРМОЕЛЕКТРИЧНИЙ ГЕНЕРАТОР 3 ПОРТАТИВНОЮ ПІЧКОЮ

У роботі наводяться результати розробки та експериментального дослідження термоелектричного генератора, що складається з термоелектричного блоку на базі армійського казанка та портативної пічки широкого використання. Отримані результати підтверджують можливість використання термоелектричного генератора для живлення акумуляторів мобільних телефонів та різноманітних гаджетів. Досягнуті енергетичні параметри суттєво переважають найближчі існуючі аналоги. Встановлено доцільність конструктивного допрацювання вибраної портативної пічки в частині забезпечення можливості використання відкритого полум'я .Економічні розрахунки пристрою визначили середню вартість ТЕГ на рівні 170 доларів США. Бібл. 7, рис. 7, табл. 2. Ключові слова: термоелектричний генератор, фізична модель, портативна пічка.

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ТЕРМОЭЛЕКТРИЧЕСКИЙ ГЕНЕРАТОР С ПОРТАТИВНОЙ ПЕЧЬЮ

В работе приводятся результаты разработки и экспериментального исследования термоэлектрического генератора, состоящего из термоэлектрического блока на базе армейского котелка и портативной печи широкого применения. Полученные результаты подтверждают возможность использования термоэлектрического генератора для питания аккумуляторов мобильных телефонов и различных гаджетов. Достигнутые энергетические параметры существенно превышают таковые, присущие ближайшим существующим аналогам. Установлена целесообразность конструктивной доработки выбранной портативной печи в части обеспечения возможности использования открытого пламени. Экономические расчеты устройства определили среднюю стоимость ТЭГ на уровне 170 долларов США. Библ. 7, рис. 7, табл. 2.

Ключевые слова: термоэлектрический генератор, физическая модель, портативная печь.

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THE USE OF THERMOELECTRIC ENERGY CONVERTERS TO REDUCE THE INFLUENCE OF NATURAL AND CLIMATIC FACTORS ON THE TECHNICAL READINESS OF A VEHICLE

The article discusses the problem associated with the operation of a vehicle at low ambient temperatures, substantiates the need for special measures to maintain the optimal thermal regime of the battery. The analysis of the factors influencing the start of a cold engine is carried out. The effect of low temperature of the storage battery on the energy performance of the electrical starting system is shown. Computational studies of the proposed system for compensating the heat losses of the storage battery during the maintenance of a vehicle at low temperatures by the method of thermostating with the use of thermoelectric energy converters are carried out. Bibl. 14, Fig. 4, Tabl. 3.

Key words: technical readiness, storage battery, thermoelectric generator, phase transition thermal accumulator, electric heating elements.

Introduction

The car has become an integral part of modern life. However, its use raises a number of problems primarily related to environmental pollution and low energy efficiency. Since the creation of the car, there has been a problem associated with ensuring a reliable and trouble-free start of the cold internal combustion engine (ICE) at low ambient temperatures. This problem is still relevant today.

The purpose of the work is to carry out computational studies of the system to ensure compensating the heat losses of the storage battery by the method of thermoelectric stabilization of its optimal temperature when a vehicle is kept out of the garage at low ambient temperatures.

Analysis of previous research

Review and analysis of literary sources related to the impact of natural and climatic factors on a vehicle during operation, primarily in urban conditions, characterized by long periods of inactivity, small movements, frequent and short stops and garage-free maintenance during the inter-shift period allows one to determine precisely the ambient air temperature as the main factor that affects the technical readiness of a vehicle.

Low temperatures complicate the start of a cold engine and lead to deterioration of its operating conditions, which generally reduces the technical readiness and use of the vehicle for its intended purpose.

The technical readiness of a vehicle at low ambient temperatures is mainly determined by the reliable start of a cold engine and the recovery time of its thermal regime. It is largely complicated as a result of a decrease in the discharge characteristics of storage battery in the mode of starting the engine due to an increase in the viscosity and resistance of the electrolyte, an increase in resistance to cranking of the engine crankshaft and deterioration of the conditions for the formation of the fuel-air mixture.

Mixing deteriorates due to a decrease in the intake temperature below the optimum, which leads to a deterioration in fuel evaporation and a decrease in the temperature of the working fluid at the end of the compression stroke. With a decrease in the ambient air temperature from $20 \degree C$ to minus $30 \degree C$, the temperature at the end of the compression stroke decreases by $100 \dots 210 \degree C$, while in diesel engines there is a delay in the autoignition of fuel two to three times in time, which leads to a deterioration of burning process. The viscosity of winter diesel fuel with a decrease in inlet air temperature from $+20 \degree C$ to minus $30 \degree C$ increases 15 times. The viscosity of gasoline when the inlet temperature decreases from $0\degree C$ to minus $30\degree C$ is one and half times higher, and evaporation is 50 percent lower.

As the temperature decreases, the viscosity of the oil in the engine lubrication system increases. This leads to an increase in friction power losses in the conjugate parts of the cylinder-piston group and as a consequence to a decrease in the cranking speed of the engine crankshaft.

The reliability of starting the internal combustion engine at low ambient temperatures is largely determined by the performance of the battery. The battery performance is understood as the maximum possible number of crankshaft rotations with a duration of 15 seconds each [1].

The decrease in the temperature of the electrolyte is accompanied by an increase in its viscosity and internal resistance, which leads to a significant decrease in voltage at the terminals of the battery, which reduces the power developed by the starter in the cold engine start mode. With a decrease in the temperature of the electrolyte from $+ 30 \degree C$ to minus $40 \degree C$, its resistivity increases 8 times [2]. According to the Research Institute of Starter Batteries, at a temperature of $0 \degree C$, the current efficiency of batteries is 90%, and at minus $40 \degree C - 20\%$. At an electrolyte temperature below minus $20 \degree C$, an intensive deterioration in the efficiency of charging batteries from the on-board network was established. When charging the battery from a stationary device, the battery electrolyte is actively boiling at a constant density. Because the energy supplied is almost completely spent on water hydrolysis, batteries are practically inoperable at minus $30 \dots 35 \degree C$ [3].

A decrease in the battery capacity in the starting mode leads to a decrease in the starting crankshaft rotation speed, and a decrease in voltage leads to a decrease in the torque developed by the starter. Achieving the required starting speed of the crankshaft at low temperatures is difficult due to an increase in the cranking resistance torque of the engine crankshaft. In the process of starting the engine at low temperatures, the determining factor is the ratio of the moment of resistance of the engine crankshaft and the torque developed by the starter.

In this connection, the main concern of ensuring the operability of the battery and, as a consequence, of the technical readiness of a vehicle as a whole, should be the maintenance of the optimal temperature of the battery. The easiest way to solve this problem is to slow down the electrolyte cooling. For example, according to the Research Institute Avtoprilad uninsulated battery 6ST-132 is cooled from + 25 ° C to minus 30 ° C at a rate of 6.6 °C for one hour; and