

Research Institute (Taiwan) and Institute of Thermoelectricity (Ukraine). The feature of this generator is its placement at some distance from the cement kiln which rotates, while it does not affect the technological processes inside the kiln. The project for waste heat recovery from garbage recycling plants using thermoelectricity was implemented jointly by Fujitaka (Japan) and Institute of Thermoelectricity (Ukraine) [104, 105].

A large number of publications are devoted to heat recovery from internal combustion engines of cars [28, 29, 52, 56 - 103] and motorcycles [28, 29]. However, it should be noted that the use of thermoelectric recuperators in cars has a number of disadvantages [60, 70, 71]. The real power gain is not significant enough. This leads to a search for more efficient applications of thermoelectricity. First of all, it looks promising to recover heat from diesel engines of large ships (in addition to high power, their advantage is the ability to remove heat from the thermoelectric converter to the surrounding water), as well as large trucks and special equipment [75, 80, 82, 93, 97]. There are also interesting works devoted to the use of thermoelectric recuperators in hybrid vehicles [71], where the energy generated during the operation of an internal combustion engine is used to recharge the batteries of the vehicle.

Ref.[106] presents the results of studies on a thermoelectric heat recuperator, which uses waste heat energy from power plants of Tokyo Electric Power. By joint efforts of the Komatsu Research Center and KELK [107], such a thermoelectric recuperator was created and its experimental studies were carried out.

In [38, 39], studies of a thermoelectric recuperator, which uses waste heat from industrial boilers, are presented. The efficiency of such a converter reaches 2%.

The Brno University of Technology (Czech Republic) has developed a thermoelectric recuperator for recovery of waste heat from a boiler, which uses biomass as a fuel [108].

The topic of heat recovery from gas turbines is discussed in [23, 110]. The exhaust gases from the turbine of pumping stations on gas mains were used as a source of thermal energy.

Refs. [111-115] present the results of development of a thermoelectric recuperator of heat from the combustion of biomass in a household kitchen stove. The temperature difference on thermoelectric modules is created on the one hand by flame, and on the other – by water tank.

One of the applications of thermoelectricity for waste heat recovery is a recuperator that uses waste heat from the biomass drying process [116]. The power generated by it is used to power the fans that circulate hot air in such a system.

Toshiba has developed a thermoelectric recuperator for an electric transformer [111].

Miniature thermoelectric recuperators used to power low-power equipment and sensors on board the aircraft are considered in [117-122]. Such devices are mounted under the wing of the aircraft and use the hot heat of the turbine.

It should be noted that heat recovery from stationary industrial plants (especially at temperatures below 600 K) is of great interest for thermoelectricity, since it allows one to fully realize its advantages. Estimates show that in the United States alone, about 3.300 TJ of energy are released annually from thousands of industrial processes [38, 53], some of which can be returned to active balance by direct thermoelectric energy conversion. Moreover, thermoelectric recuperators can be used not only to increase the overall efficiency of energy conversion, but also to provide backup power to the most critical units of industrial installations, which can significantly increase their reliability [23-25].

Determination of general properties of thermoelectric recuperators

Physical model of a thermoelectric sectional heat recuperator is shown in Fig.2.

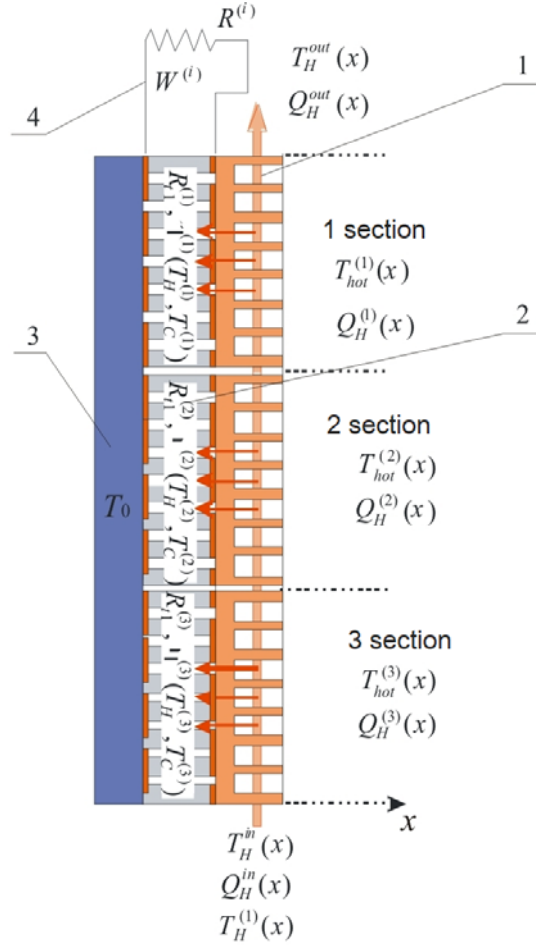


Fig. 2. Physical model of thermoelectric sectional heat recuperator:
1 – hot heat exchanger; 2 – thermopiles
3 – cold heat exchanger 4 – matched electrical load of the section.

Each section of the recuperator consists of hot heat exchanger (1), thermopile (2) with thermal resistance $R_{t2}^{(i)}$ and efficiency $\eta(T_H, T_0)$; cold heat exchanger (3) with temperature T_0 . Thermopiles of each recuperator section are loaded on matched electrical load $R^{(i)}$ (4). Hot gas input flow has temperature T_H^{in} and thermal power Q_H^{in} . Hot gas gives part of heat $Q_H^{(i)}(x)$ at temperature $T_{hot}^{(i)}(x)$ to the hot heat exchanger. At the recuperator outlet the gas flow has temperature T_H^{out} and thermal power Q_H^{out} . From the hot heat exchanger heat is passed to the thermopile, heating its hot side to temperature $T_H^{(i)}(x)$. To calculate maximum possible recuperator power, we will ignore thermal losses. For the optimization of TEG it is necessary to find the distribution of temperatures and heat flows in thermopiles of each section. Such a calculation for this model was made through use of numerical computer methods.

To calculate the electrical power of TEG, we use the equation of energy balance in the form

$$W = \sum_{i=1}^N \left[\int (\mathcal{Q}_H^{(i)}(x) - \mathcal{Q}_C^{(i)}(x)) dx \right]. \quad (1)$$

The necessary temperatures and heat flows are found from thermal conductivity equation

$$-\nabla(\kappa_{TE}(T)\nabla T) = Q_J, \quad (2)$$

where κ_{TE} is effective thermal conductivity of thermopile, Q_J is the Joule heat which is released in the bulk of the thermopile.

The boundary conditions for (2) will be given by

$$Q_H^{in(1)} = Q_H^{in}, \quad Q_H^{in(i+1)} = Q_H^{out(i)}, \quad Q_H^{out(N)} = Q_H^{out}, \quad (3)$$

$$Q_H^{(i)}(x) = (T_H^{(i)}(x) - T^{(i)}(x)) / R_t^{(i)}, \quad (4)$$

$$Q_C^{(i)}(x) = (T_0(x) - T^{(i)}(x)) / R_{t2}^{(i)}, \quad (5)$$

The set of relations (1) - (5) makes it possible to determine the distribution of temperatures and heat flows in each of the sections

To restrict the hot temperature of the module, the thermal resistance between the hot heat exchanger and the thermoelectric module is determined from equation (4).

The power of each section and total efficiency of TEG can be found from equations

$$W^{(i)} = \int Q_H^{(i)}(x) \eta(T_H^{(i)}(x), T_0) dx, \quad (6)$$

$$\eta_{TEG} = \frac{1}{Q_H^{in}} \sum_{i=1}^N W^{(i)}. \quad (7)$$

The system of equations (1) - (5) was solved by numerical methods on a two-dimensional mesh of finite elements.

To calculate the efficiency of the thermoelectric recuperator, thermoelectric materials based on Bi-Te were selected which are one of the best in terms of quality in the considered temperature range [127].

At the first stage, the optimization of the hot temperatures of the recuperator sections was carried out (Fig. 3). Fig. 4 shows the relative number of thermoelectric modules of the same type in a section to achieve optimal temperature distribution.

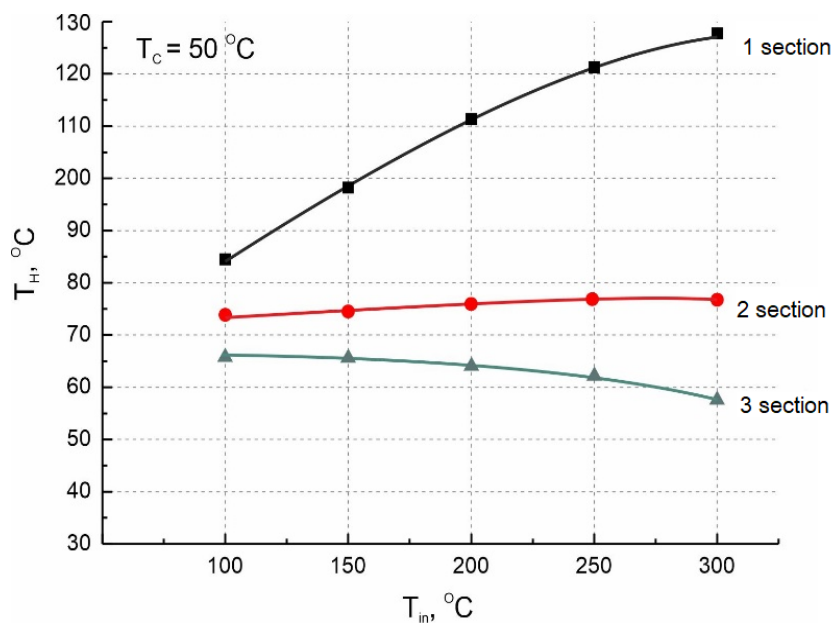


Fig. 3. Dependence of the optimal hot temperature of the sections on the temperature of heat carrier.

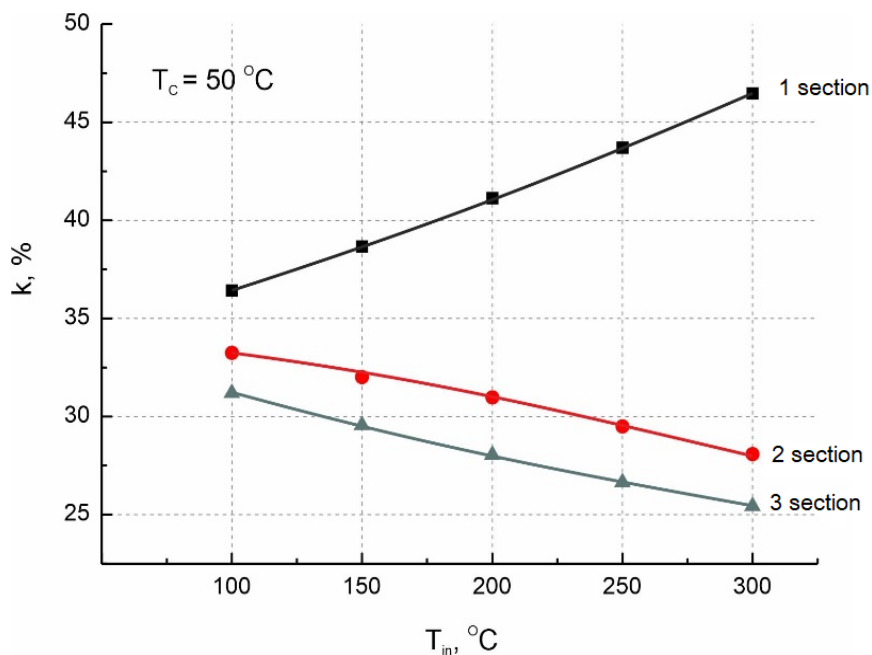


Fig. 4. Relative number of thermoelectric modules in a section to achieve optimal temperature distribution.

The next step is to determine the dependence of the efficiency of thermoelectric modules (Fig. 5) and the recuperator as a whole (Fig. 6) on the temperatures of the input heat carrier.

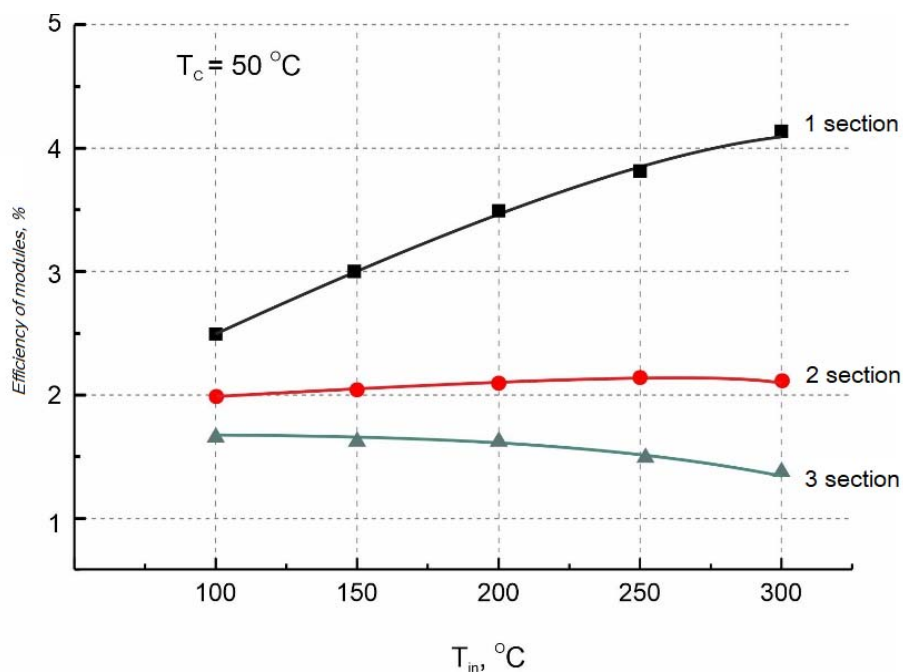


Fig. 5. Dependence of the efficiency of modules of sections on the temperature of inlet heat carrier.

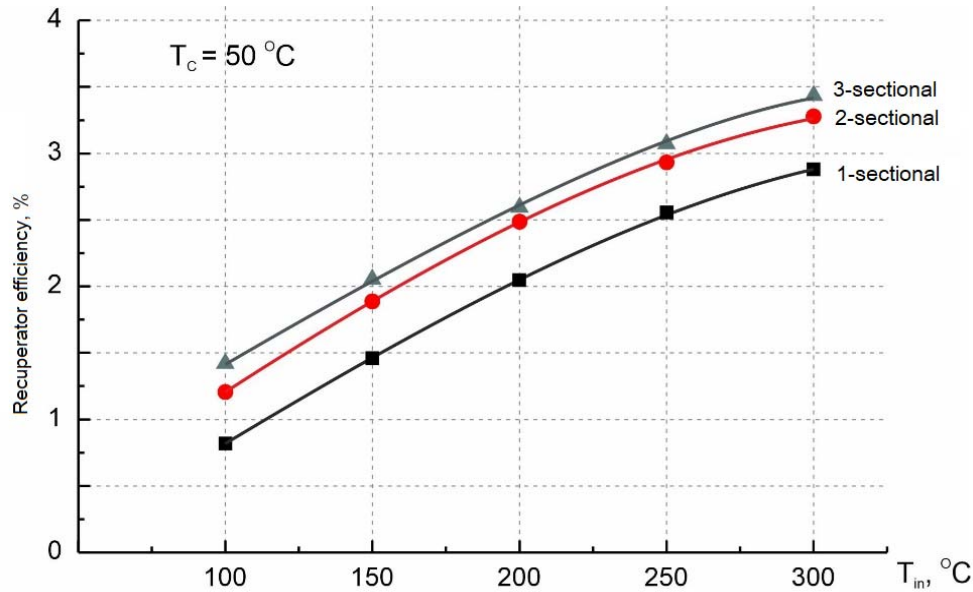


Fig. 6. Dependence of recuperator efficiency on the temperature of inlet heat carrier.

As is evident from Fig. 6, the use of the second section of the thermoelectric heat recuperator leads to an increase in efficiency by ~18%, and the third – only by 3%.

The percentage contribution of each section of thermoelectric heat recuperator to its total power is shown in Fig. 8. As can be seen from the figure, the percentage contribution of the first section of recuperator to total power is 85 – 90%, the second – 8 – 12%, the third – about 2%.

To assess the economic feasibility of using a thermoelectric recuperator, its specific cost was calculated (Fig. 8), based on the results obtained in [128]. As can be seen from the figure, the use of the third section in the considered temperature range is not economically feasible. The use of the second section is also questionable.

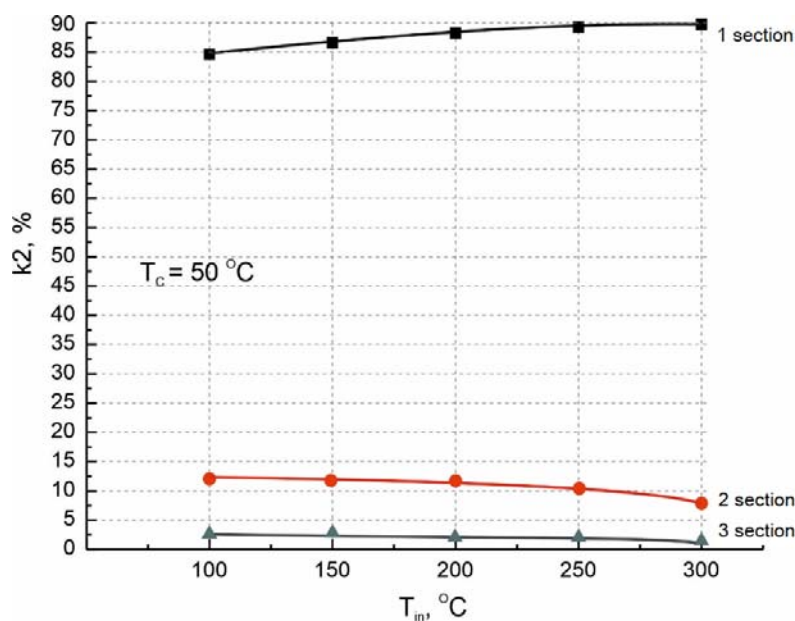


Fig. 7. Contribution of each section to total power of recuperator.

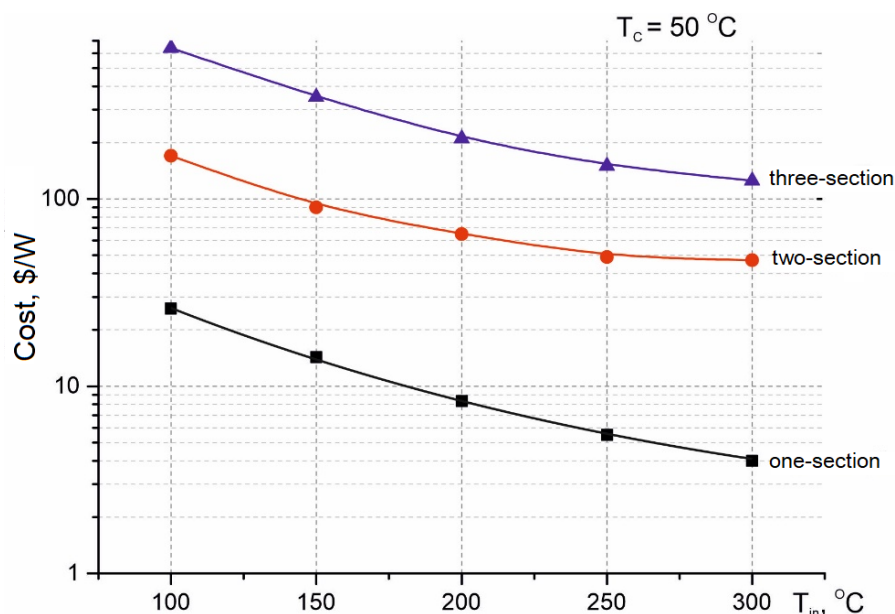


Fig. 8. Specific cost of sectional recuperators.

For a better understanding of the economic efficiency of thermoelectric recuperators, we will calculate their payback time, based on a comparison of the cost of their electrical energy with the cost of industrial electrical energy. Fig. 9 shows the results of such calculations. For example, a comparison was made with the average cost of electricity in Ukraine 0.12 \$ / (kWh) (according to the data of the company Ukrenergo [130]).

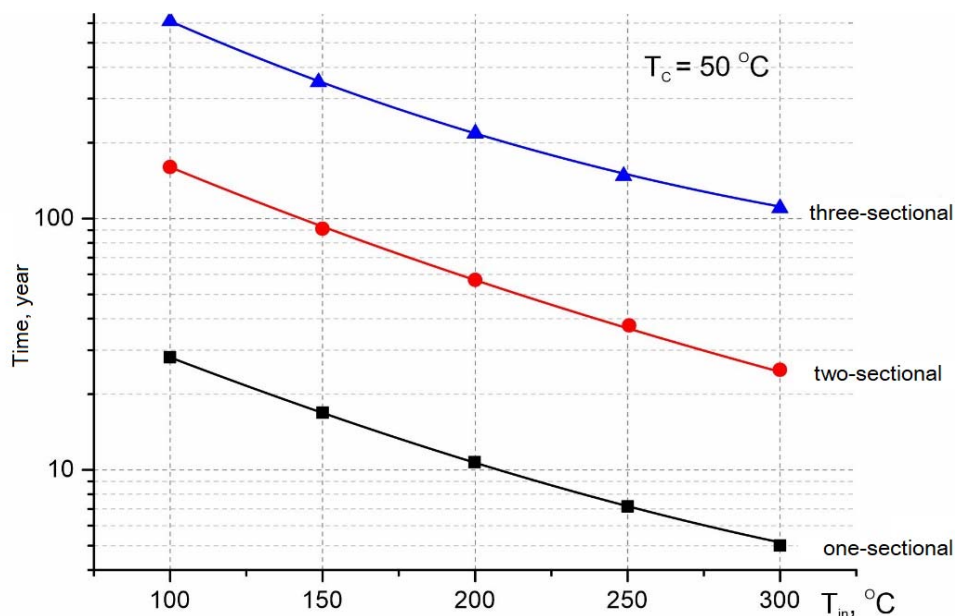


Fig. 9. Payback time of sectional recuperators.

From the analysis of Fig. 9 it becomes clear that for the specified temperature range (100 - 300 °C) it is economically feasible to use only one section. A small gain in power when using other sections does not cover material costs.

Conclusions

1. The dependences of the optimal temperatures of the recuperator sections on the inlet gas temperature in the range from 100 to 600 °C are established. For the first section from 37 to 47 °C, the second - from 33 to 27 °C, the third - from 32 to 25 °C.
2. The number of thermoelectric converters in each section is determined to achieve the optimal temperature distribution in the sections. For low inlet gas temperatures the number of thermocouples in the sections is approximately the same. With a rise in temperature, the share of thermocouples in the first section increases.
3. The specific cost of a thermoelectric recuperator and its payback time are calculated. It is shown that the cost of each subsequent section is approximately an order of magnitude greater than the cost of the previous one. Therefore, for the range of hot temperatures of the heat carrier 100-300°C it is economically feasible to use only one section.

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Анатичук Л.І., акад. НАН України^{1,2}
Кузь Р.В., канд. фіз.-мат. наук^{1,2}

¹Інститут термоелектрики НАН і МОН України,
вул. Науки, 1, Чернівці, 58029, Україна;
e-mail: anatyshuk@gmail.com;

²Чернівецький національний університет
ім. Юрія Федьковича, вул. Коцюбинського 2,
Чернівці, 58000, Україна

ЕФЕКТИВНІСТЬ ТЕРМОЕЛЕКТРИЧНИХ РЕКУПЕРАТОРІВ ДЛЯ РАЦІОНАЛЬНИХ ТЕМПЕРАТУР ДЖЕРЕЛ ТЕПЛА

У роботі наводяться результати аналізу термоелектричних рекуператорів теплових відходів для діапазону температур теплоносія 100 -300°C. На основі комп'ютерної моделі проведено оптимізацію секційних рекуператорів, розраховано ККД кожної секції та рекуператора в цілому. Розраховано питому вартість та час окупності секційних генераторів. Зроблено висновки про економічну доцільність використання таких рекуператорів. Бібл. 130, рис. 9, табл. 1.

Ключові слова: термоелектричний рекуператор, відпрацьоване тепло, ККД, потужність, питома вартість.

Анатичук Л.І., акад. НАН України^{1,2}

Кузь Р.В., канд. физ.-мат. наук^{1,2}

¹Институт термоэлектричества НАН и МОН Украины,
ул. Науки, 1, Черновцы, 58029, Украина, e-mail: anatych@gmail.com;

²Черновицкий национальный университет
им. Юрия Федьковича, ул. Коцюбинского, 2,
Черновцы, 58012, Украина

ЭФФЕКТИВНОСТЬ ТЕРМОЭЛЕКТРИЧЕСКИХ РЕКУПЕРАТОРОВ ДЛЯ РАЦИОНАЛЬНЫХ ТЕМПЕРАТУР ИСТОЧНИКОВ ТЕПЛА

В работе приводятся результаты анализа термоэлектрических рекуператоров тепловых отходов для диапазона температур теплоносителя 100 -300⁰С. На основе компьютерной модели проведена оптимизация секционных рекуператоров, рассчитан КПД каждой секции и рекуператора в целом. Рассчитаны удельная стоимость и время окупаемости секционных генераторов. Сделаны выводы об экономической целесообразности использования таких рекуператоров. Библ. 130, рис. 9, табл. 1.

Ключевые слова: термоэлектрический рекуператор, отработанное тепло, КПД, мощность, удельная стоимость.

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